

Understanding "Heat Load & Psychometric"

by Djunaidi HS







What is the heat load? ⇒ It refers to the cooling and heating loads.

What is the heat load calculation? ⇒ It is the calculation of the cooling and heating loads.

Usually software is used to do the calculations.

The calculation results are used to determine the capacity of air conditioners.







Understanding Of Heat Load (Human Comfort)



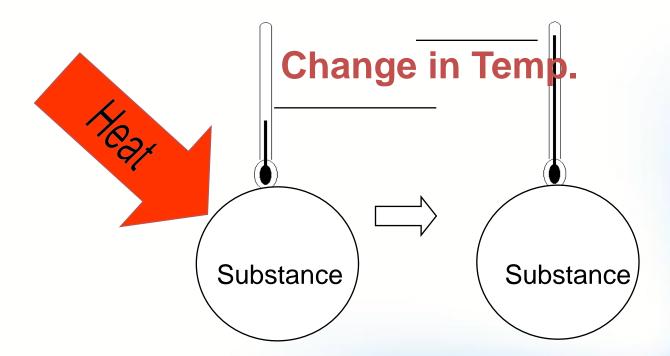
1) Heat Load

- Classified into 2 main types:

- 1. Sensible Heat
- 2. Latent Heat



Sensible Heat

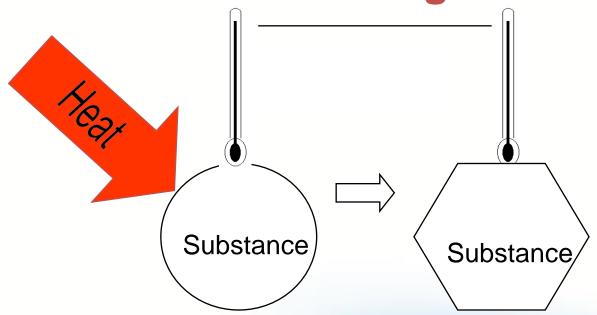


(No change in physical state)



Latent Heat

No change in temperature



(Physical state is changed)



1.2) Classification of Cooling Loads

- 1) Skin Loads
- 2) Internal Loads
- 3) Other Loads

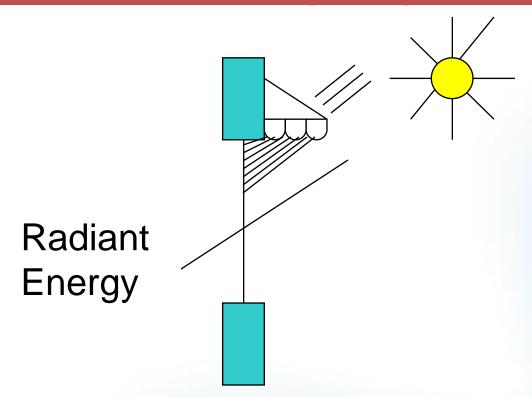


1) Skin Loads

Skin loads originate from the heat sources outside or external to the conditioned space



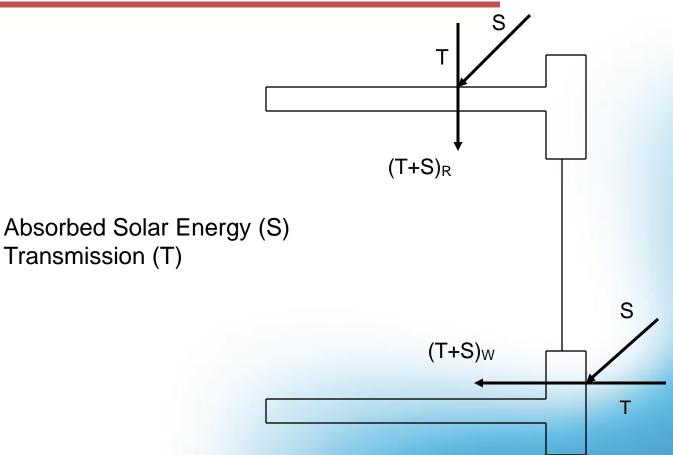
a) Solar Gain Through glass (SG).



Solar heat gain is reduced by the use of internal or external shading devices such as overhang.

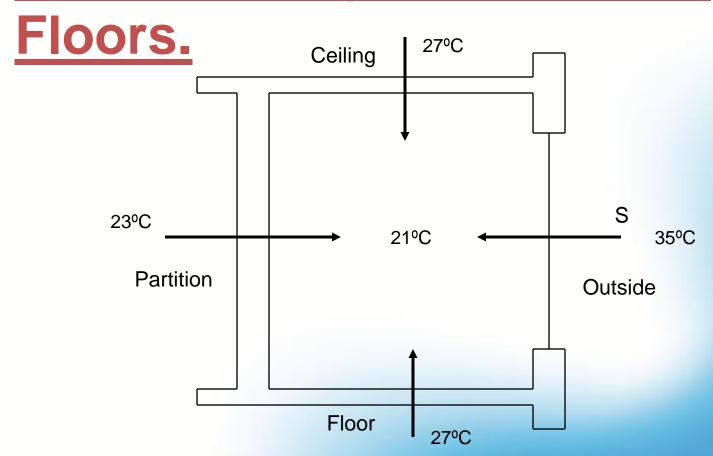


b) Solar & Transmission Gain Thru Walls and Roofs.

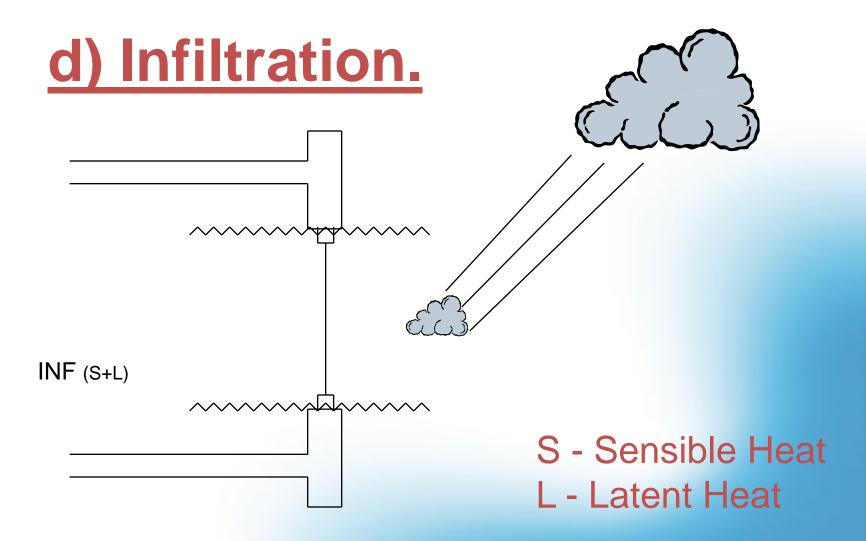




c) Transmission Thru Glass, Ceiling, Partitions or

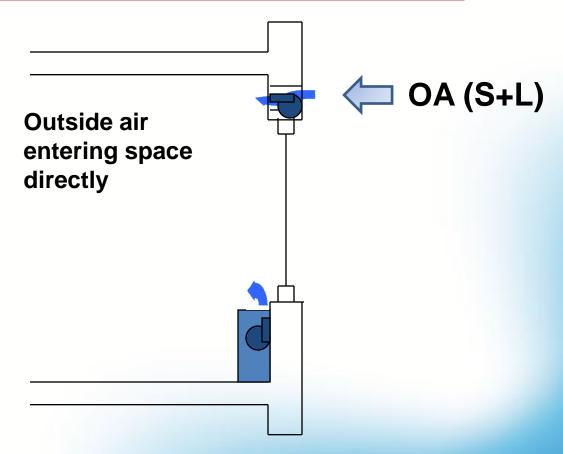








e) Ventilation (O.A)





Skin Loads

- a) Solar Gain Through glass (SG).
- b) Solar and Transmission Gain Through Walls and Roofs.
- c) Transmission Through Glass, Ceiling, Partitions or Floors.
- d) Infiltration.
- e) Ventilation (O.A)

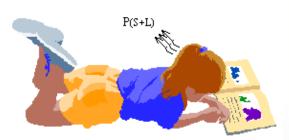


2) Internal Loads

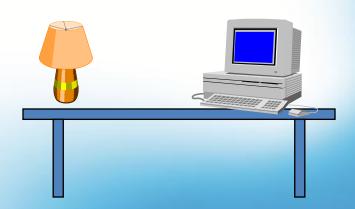


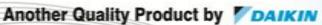
a)Lighting Loads

b)People Loads



c)Equipment Loads







Room Loads

- · Add all sensible loads together results in sensible heat gain.
- · Add all latent load together results in latent heat gain.

Sum of the Room Sensible Heat and the Room Latent Heat Is the Room Total Heat



3) Other Loads

a. Supply Air Side



c. Outside or Ventilation Air

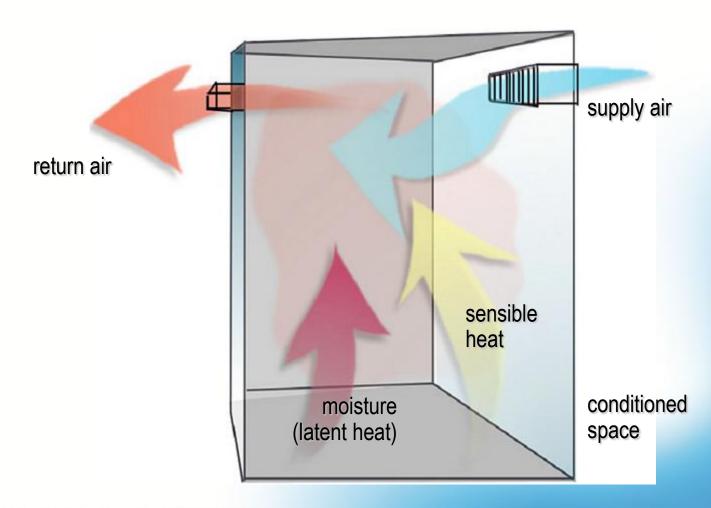






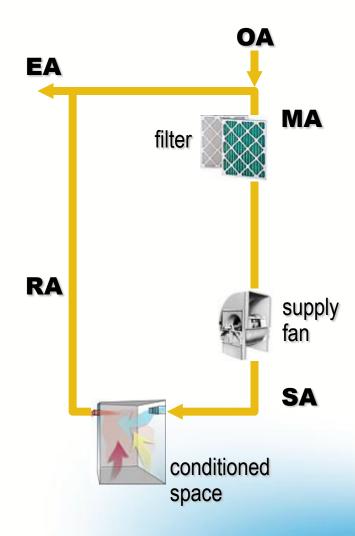


Airside Loop



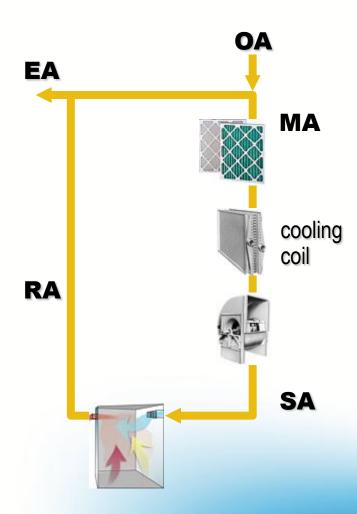


Supply Fan and Filter





Cooling Coil



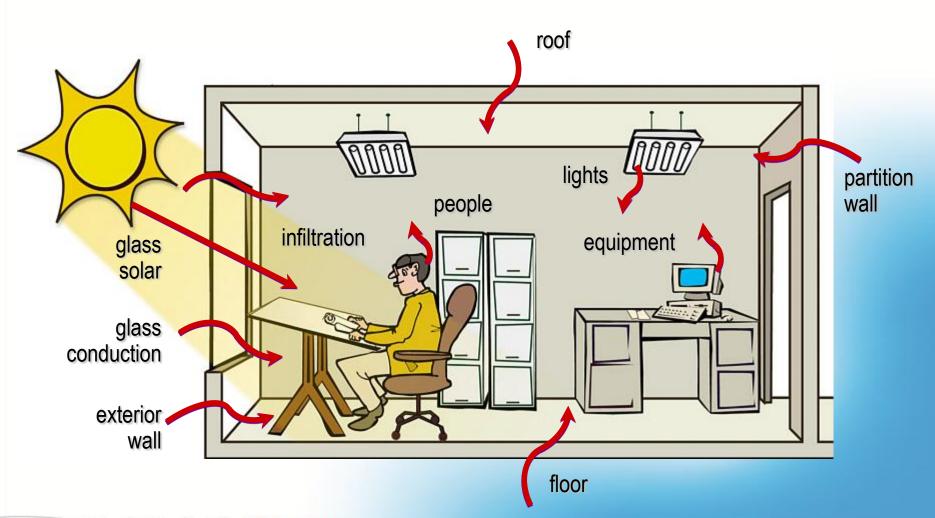
Cooling Load Components



1.Radiation

2.Conduction

3.Convection





Cooling Load Components

cooling load components	sensible load	latent load	space load	coil Ioad
conduction through roof, walls, windows, and skylights	✓		✓	V
solar radiation through windows, skylights	S 🗸		V	V
conduction through ceiling, interior partition walls, and floor	✓		✓	V
people	V	V	V	✓
lights	\checkmark		\checkmark	✓
equipment/appliances	V	V	V	✓
infiltration	V	\	V	✓
ventilation	V	V		✓
system heat gains	V			V





WHY Heat Load Calculation? Cooling Load Estimation



Recommended IAQ (SS 554:2009)

By law, the indoor conditions of an air con space shall maintained within the following limits:

Design Temp. : 25°C +/- 1 °C

Relative Humidity: < 65% (for new buildings)

< 70% (for existing buildings)

Air movement : 0.10 ~ 0.30 m/s

Note: At occupant level of 1.5m above floor.



Heat Transfer Coefficient (U-value)

Defined as the quantity of heat transmitted under steady state conditions through unit area of the material of unit time when the temperature difference exists between it opposite surfaces.

(W/m ^{2 o}C)



Calculation of U- value

(Coefficient of heat transmission of wall structure)

$$1/U = 1/\alpha_0 + I_1/\lambda_1 + I_2/\lambda_2 + \dots + I_n/\lambda_n + R + 1/\alpha_1$$

α₀: External surface heat transfer coefficient [23] [W/m² °C]

α₁: Internal surface heat transfer coefficient [9] [W/m² °C]

I : Thickness of material [m]

λ : Coefficient of thermal conductivity of material [W/m °C]

R: Thermal resistance of air layer [m² °C/W]



Exercise: Calculate U value of outside wall

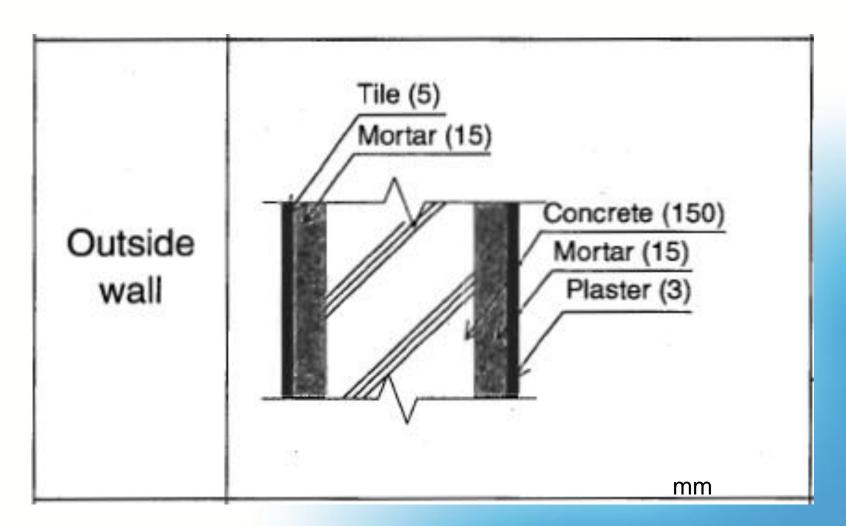




Table 4-11 External surface heat transfer coefficient, αο [W/m².°C]

Location of surface	Summertime	Wintertime
Rectangular exterior wall surface	17	23
Rooftop surface	23	35
Planceer	17	17

Table 4-12 Internal surface heat transfer coefficient, αi [W/m².°C]

Location of surface	Direction of heat flow		
	Upward	Downward	
Horizontal	9.26	6.13	
Inclined	9.08	7.49	
Vertical	Horizontal	8.29	



Table 4-13 Coefficient of Thermal Conductivity of Materials, λ [W/m².°C]

Name of material	Coefficient of thermal conductivity λ [W/m²· 'C]	[W/m.ºC] Name of material	Coefficient of thermal conductivity λ [W/m²·°C]
Air (static)	0.022 _{[W/c}	m.ºC] alt and the like	[W/m.ºC]
Water (static)	0.6	Moisture-proof paper and the like	0.21
Ice	2.2	Tatami mat	0.15
Snow	0.06	Composite tatami mat	0.07
Steel	45	Carpet and the like	0.08
Aluminum	210	Wood (heavy-weight)	0.19
Copper	390	Wood (medium-weight)	0.17
Rock (heavy-weight)	3.10	Wood (light-weight)	0.14
Rock (light-weight)	1.40	Plywood	0.19
Soil (argilliferous)	1.50	Soft fiber board	0.056
Soil (arenaceous)	0.90	Semi-soft fiber board	0.14
Gravel	0.62	Hard fiber board	0.22



PC concrete	1.50	Particle board	0.17
Plain concrete	1.40	Wood wool cement board	0.19
Light-weight concrete	0.78	Glass wool (24K)	0.042
Autoclaved lightweight concrete (ALC)	0.17	Glass wool (32K)	0.040
Concrete block (heavy-weight)	1.10	Rock wool heat insulating material	0.042
Concrete block (light-weight)	0.53	Sprayed rock wool	0.051
Mortal	1.50	Rock wool acoustic board	0.064
Asbestos slate	1.20	Polystyrene foam board (bead)	0.047
Plaster	0.79	Polystyrene foam board (extruded)	0.037
Plaster board / Lath board	0.17	Polystyrene foam board (Freon molded)	0.026
Grout	0.74	Rigid urethane foam board	0.028
Mud wall	0.69	Soft urethane foam board	0.050
Glass	0.78	Polyethylene foam board	0.044
Tile	1.30	Rigid PVC foam board	0.036
Brick wall	0.64	Airtight hollow layer, R = 0.15 m ² . °C/W	
Roofing tile	1.00	Non- airtight hollow layer, R = 0.07 m ² .*C/W	
Synthetic resin / Linoleum	0.19		
FRP	0.26		



Answer:

$$\frac{1}{U} = \frac{1}{23} + \frac{0.005}{1.3} + \frac{0.015}{1.5} + \frac{0.15}{1.4} + \frac{0.015}{1.5} + \frac{0.003}{0.79} + \frac{1}{9} = 0.289$$

$$U = 3.46 \text{ W/m}^2 \, ^{\circ}\text{C}$$





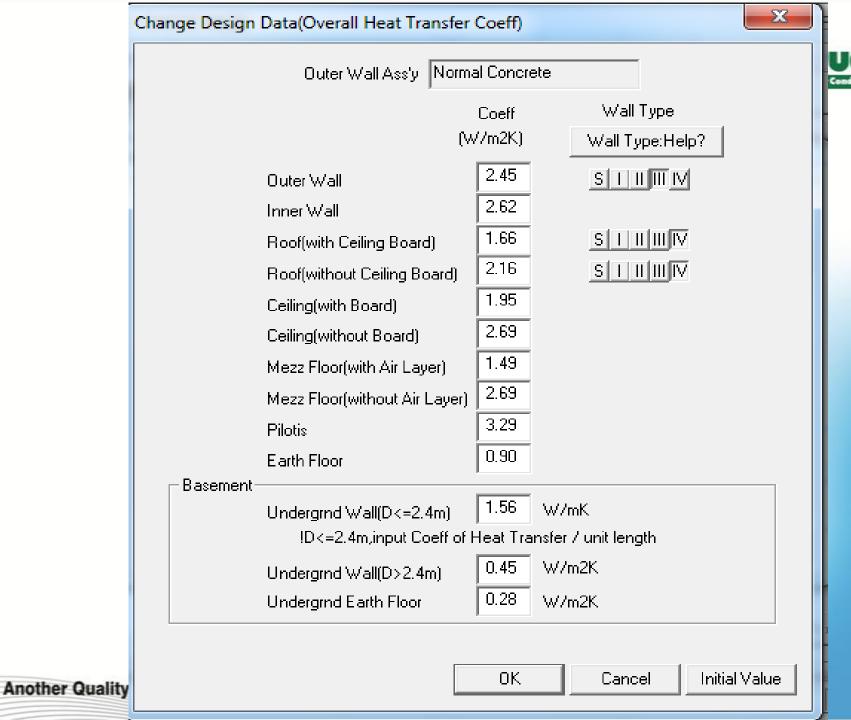


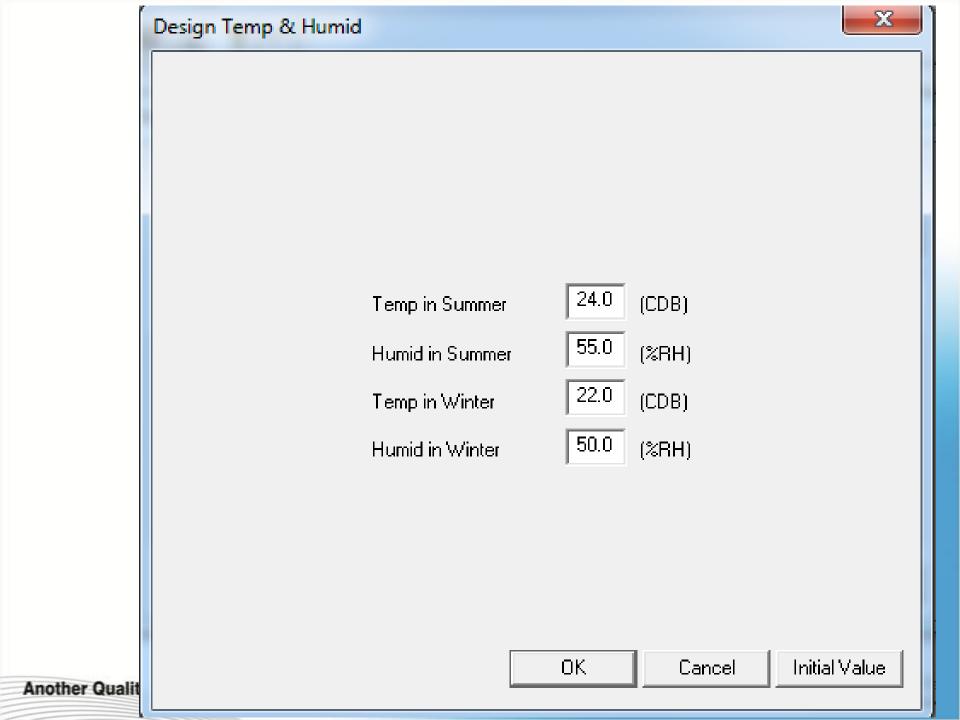


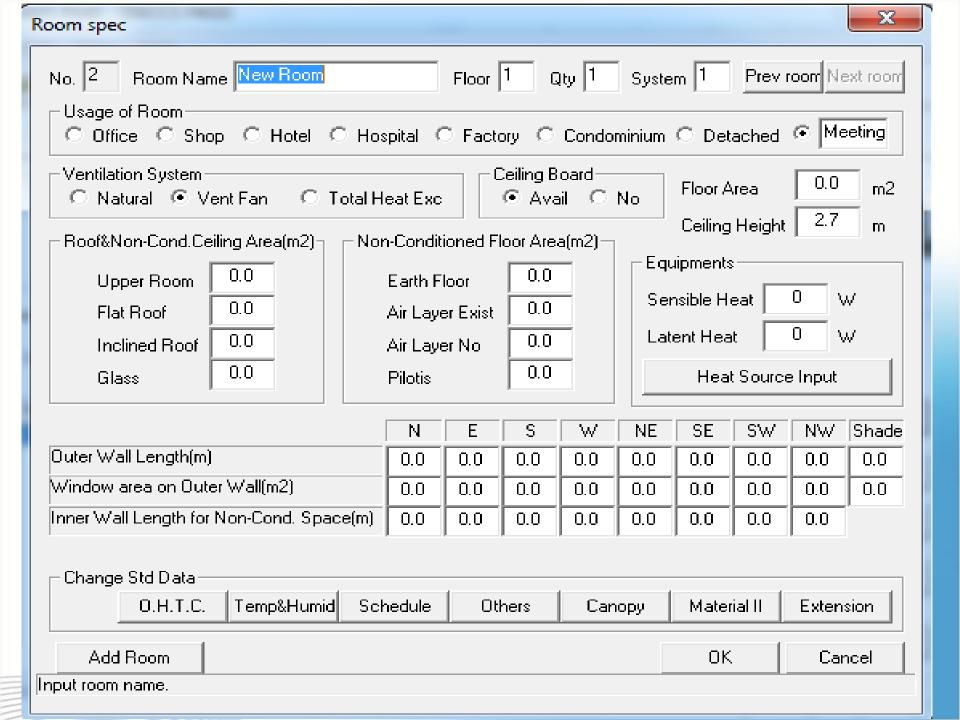
Heat Load Calculation HKGSG Shortcut



Project Outline		X
Project Name	ABC	
City/Country	Jakarta/Indonesia	City/Country Map
Address	123Jakrta	
Cuter Wall As:	semblies ————	
Nomal Concrete ALC Plate Metallic Curtain Wall Wooden construction Others		Init value of overall heat trans coeff of main part Roof(with Ceiling Board) : 1.66 W/m2K Outer Wall : 2.72 W/m2K Inner Wall : 2.62 W/m2K
		OK Cancel Design Data







X Schedule New Room Room Name Operating Time Zone Set Schedule |18| Нг Hr to Hour 100 100 100 100 Lighting Persons 100 100 |100|100| |100 |100 |100 |100 |100 Equipments Hour |100||100| 100 100 Lighting |100||100| 100 100 Persons 100 l Equipments 100 | 100 | 100 | 100 | 100 | 100 | OK. Cancel Initial Value

Another Q

Anothe

Input window and Canopy



Please select direction of window with

If there is window without canopy in the same direction, please input zero for canopy dimensions.

										[m].
Direction	Dc	DI	Dr	Al	Ww	Αr	В	Hw	Нс	`
N v	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	
N 🔻	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	
N 🔻	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	
N 🔻	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	
N 🔻	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	
N 🔻	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	
N 🔻	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	
N 🔻	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	
N 🔻	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	
N 🔻	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	

0	
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Qty

OK Cancel

Material II New Room Room Name 0.H.T.C.(W/m2K): Window: Window Type Clear 5mm ₹ Wall Type 0.00 S I II III IV Outer Wall Neutral color Blind Type 0.00 Inner Wall Shading Factor: 0.63 O.H.T.C.: 4.97 SE NWN Ε S NE SW. Shade W 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 Outer Wall Length(m) Window Area on Outer Wall(m2) 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 Inner Wall Length for Non-Cond.Space(m) 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 OK. Cancel



Room Name New Room

!Detail Set Room Data. Nomally no need to change.

Personnel Heat Gain per Person-

Sensible Heat 56 W/person Latent Heat 76 W/person

-Glass Surface Ratio(without sash area):

Glass Area = Ttl Window Area $^* | 0.95 |$!Use for Solar heat gain through

Next Room Condotion

Next room temp diff is calculated using below factor.

(dt)=(O/D Temp-I/D Temp) * Temp Diff Coeff

				_							
Coeff	Cooling	0.4	0.4	0.4	0.4	0.4	0.4	0.4	0.4	0.4	0.4
Temp Diff	Heating	0.4	0.4	0.4	0.4	0.4	0.4	0.4	0.4	0.4	0.4
Boiler/Kito in Next F	No	No	No	No	No	No	No	No	No	No	

ОК

NE

Cancel

SE SW NW Deiling Floor

Initial Value



Aircon heat load calculation sheet

Project Name: Kesuma Resort, Bali

Address: Samabe, Bali

13/September/2012

Heat load sum up table

(Upper:W, Lower:kcal/h)

			0		Cooli	ng			Heating			Floor	Heat load	i per area
Room name	Fl	Sys -tem	Qty. of rooms	Indoor SH	Total	Selected	Time	Total	Selected	Humid.	Time	area	Cooling	Heating
			rooms		[W] (kcal/	'h)	[Hr]	[W] (kcal/h)		[kg/h]	[Hr]	[m2]	[W/m2] (kc	:a1/hm2)
Living Room 1	1	1	1	5114 (4398)	6744 (5800)	7081 (6090)	16	7 (6)	(7)	-0.99	6	25.3	279.9 (240.7)	0.3
Living Room 2	1	1	1	2590 (2227)	3564 (3065)	3742 (3218)	17	(3)	(4)	-0.61	6	16.8	222.8 (191.6)	(0.2)
Entrance	1	1	1	2088 (1796)	3053 (2626)	3206 (2757)	17	(3)	(4)	-0.59	6	15.3	209.5 (180.2)	(0.2)
Peak load of but	ildin	ıg	3	9663 (8310)	13260 (11404)	13923 (11974)	16	15 (13)	17	-2.19	6	57.4	242.6 (208.6)	0.3

SH : Sensible heat

Table of system heat load (Upper:W, Lower:kcal/h)

				Cooling							Heatin	g		
Sys -tem	Time	F/A vol	Indoor SH	Indoor	Outside	Total	Selected	Time	F/A vol	Indoor	Outside	Total	Selected	Humid.
	[Hr]	[m3/h]			[W] (k	cal/h)	•	[Hr]	[m3/h]		[W]	(kcal/h)		[kg/h]
1	[Hr]	[m3/h] 220	9663 (8310)	10968 (9432)	[W] (h	13260 (11404)	13923 (11974)	[Hr]	[m3/h] 220	(0)	[W] 15 (13)	(kcal/h) 15 (13)	(17 (14)	[kg/h] -2.19

If total indoor heat load is negative, it is not calculated.

F/A : Fresh air SH : Sensible heat

System table

System	Room name	Floor	Rooms
1 1 1	Living Room 1 Living Room 2 Entrance	1 1 1	1 1 1

System	Room name	Floor	Rooms

Room data(Input data)

Project name	Kesuma Resort, Bali	Design room temperature in summer(CDS) 26.0 Design room humidity in summer(\$RS) 50.0
Address	Sanabe, Bali	Design room temperature in winter(CDB) 22.0 Design room humidity in winter(4RB) 50.0
City	Den PaSar/Indonesia	
Outer wall assembly	Normal Concrete	Fresh air intake Air volume(m3/h person) Summer 20.0
Max. fresh air temp. in summer(C)	31.0	Winter 20.0
Min. fresh air temp. in winter(C)	21.9	infiltration wentilation(Times/h) Summer 0.20 Winter 0.30
Room name	Living Room 1	Heating load internal heat gain[to ratio of cooling load internal heat gain](%) Lighting: 50 Fermons: 50 Equipments:
Floor No	1	
System No	1	Window tube cl> Heat absorbing Snm. Blind type Heutral tints Shading factor/ONTC 0.56/4.97
No of rooms	1	
Usage of Room	Hotel	
Ceiling board	Available	Lighting Fluorescent lamp(N/m2) 30.0 Incandescent lamp(N/m2) 0.0
Method of fresh air intake	Common ventilation fan	No of persons 5
Floor area(m2)	25.3	7
Ceiling height(m)	3.6	Depth of underground wall(m) 0.0 fUnderground wall is valid only when outer wall is negative value. Height above attic(m) 0.6
RoofsMon-air-conditione ceiling area(m2)	1 Overhead room, Flat roof , Inclined roof , Window glass 0.0 0.0 25.3	Humidifying method Without humidifier
Non-air-conditioned floor area(m2)	Earth floor , with air layer, Without air layer, Filotis 0.0 5.6 0.0 0.0	Overall heat transfer coefficient (W/m2K) Wall type Outer wall <pre>ci></pre>
Outer wall length(m) <1	N R S N NE SE SW MN Stude	Inner Wall <1> 2.62
	0.0 0.0 0.0 0.0 0.0 8.4 8.7 0.0 0.0	Roof(with ceiling board) 1.66 IV Ceiling(with ceiling board) 1.95
outer wall(m2) Non-conditioned innerch	0.0 0.0 0.0 0.0 0.0 2.7 5.5 0.0	Mezz floor(with air layer) 1.49 Mezz floor(without air layer) 2.69
wall length(n)	ength with negative value is regarded as underground wall.	Pilotis 3.29 Earth floor 0.90
	Sensible heat, Latent heat	Underground wall(depth<-2.4m) 1.56 (W/mK) Underground wall(depth>2.4m) 0.45
from equipments(W)	500 0	
		Safety factor Cooling 1.05 Heating 1.10
Operating time Tone	4:00 to 24:00	
Internal heatgaing sche	fule(%) Time 4 5 6 7 8 9 10 11 12 13 14 15 16 Lighting 100 100 100 100 100 100 100 100 100 10	
	Persons 100 100 100 100 100 100 100 100 100 10	100 100 100 100 100 100 100 100

Table of room heat load

Room name	Floor	System	Rooms	Usage	F1 area(m2)	Height (m)	No of person	F/A volume (m3/h)
Living Room 1	1	1	1	Rotel	25.3	3.6	5	Summer 100.0/Winter 100.0

[Cooling load] Condition of indoor design temprature & humidity: 26.0(CDB) 50.0(RRH) [W] (kcal/h)

	9	F/A	cond.	Outer	Roof	Inner	Floor	Window	Infilt	ration	Runan	body	Light	Equip	ments	Indoor	heat	Fresh	Total heat los	
		CDB	BRI	Wall	Ceiling	Wall	22001	*11110#	SH	IH	SH	LR	-ing	SH	LH	SH	LH	air	Total Heat In	814
•	4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 22 23 24	27.1 27.0 27.2 27.2 27.7 28.5 29.4 31.1 31.7 31.0 31.7 31.3 30.6 29.9 29.1 28.5 28.1 27.6 27.4	08.0 09.3 09.3 08.7 06.6 03.4 79.6 75.0 70.4 70.0 75.0 75.0 77.7 00.9 03.4 05.0 06.1 07.7	45 35 35 28 40 86 154 228 304 368 425 471 507 542 559 549 495 415 325 239 167	23 10 10 14 14 14 20 74 140 259 393 541 670 753 701 740 656 527 300 263 162 92	34 31 37 52 77 106 136 150 176 176 142 121 96 77 65 55 49 43	4 3 3 4 6 0 11 15 17 19 19 19 10 15 13 10 6 5 5	94 05 05 1125 1521 1600 1557 1502 1625 1637 1635 1043 1965 1726 927 263 212 179 153 136 119	7 6 6 7 10 15 20 26 31 34 35 32 20 23 19 15 10 0	147 147 149 150 153 153 150 150 150 150 150 150 150 150 150 150	295 295 295 295 295 295 295 295 295 295	430 430 430 430 430 430 430 430 430 430	880 880 880 880 880 880 880 880 880 880	500 500 500 500 500 500 500 500 500 500	000000000000000000	1882 1853 1853 2890 3318 3577 3730 4069 4322 4509 4888 5114 4909 4066 3268 3009 2742 2488 2276 2109	577 577 579 500 500 500 500 500 500 500 500 500 50	945 942 957 992 925 962 1012 1035 1055 1055 1042 1019 900 952 925 903 863	3304 (204) 3272 (201) 3272 (201) 4326 (372) 4700 (411) 5005 (437) 5144 (442) 5330 (456) 5692 (409) 5965 (513) 6155 (563) 6744 (500) 6516 (560) 5640 (405) 4005 (413) 4517 (300) 4227 (363) 3953 (340) 3735 (321) 3551 (305)	40 40 00 11) 30 40 40 50 00 20 50 50 50 20 20 20 20 20 20 20 20 20 20 20 20 20

F/A : Fresh air SH : Sensible heat LH : Latent heat

(bca1/h)

[Heating load] Condition of indoor design temprature & humidity: 22.0(CDB)

50.0 (ERR)

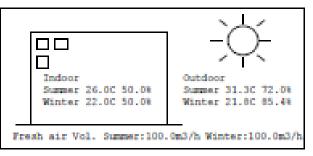
	m./s			David.	-			7-8114			See dec		2-4-		Testere				
Time	F/A	cond.	Outer	Roof	Inner	Floor	Window	Infilt	ration	NUMBER	body	Light	Equip	ments	Indoo	Lest	Fresh	Total hea	t load
	CDB	ł RH	Wall	Ceiling	Wall			SH	IH	Sil	н	-ing	SH	LH	SH	LH	air		
4	22.2	03.9	0	268	0	0	0	0		-140	0	-440	-250	-0	0	0	0	0 (0)
5	22.0	04.3	0	277	0	0	0	0	0	-140	0	-440	-250	-0	0	0	0	0 (0)
* 6	21.0	05.4	15	206	6	1	19	2	0	-140	0	-440	-250	-0	0	0	7	7 (6)
7	22.0	04.3	0	277	0	0	0	0	0	-140	0	-440	-250	-0	0	0	0	0 (0)
0	22.6	02.4	0	0	0	0	0	0	0	-140	0	-440	-250	-0	0	0	0	0 (0)
9	23.4	79.6	0	0	0	0	0	0	0	-140	0	-440	-250	-0	0	0	0	0 (0)
10	24.4	76.5	0	0	0		0	ų.	0	-140	0	-440	-250	-0				0 (0)
11	25.4	73.0 70.1	0	0	0	0	0	0	0	-140 -140	0	-440 -440	-250	-0 -0	0	0	0	0 (0) 0)
**	26.3				_		-		_				-250	-			0	0 (
13 14	26.0 27.0	68.5 67.7	0	0	0		0		0	-140 -140	0	-440 -440	-250 -250	-0 -0			0	0(0) 0)
15	26.9	60.1	0	0	0		0		0	-140	0	-440	-250	-0	0		0	0 (
16	26.7	69.0	ő	0	0	0	ň	ŏ	ő	-140	0	-440	-250	-0	ă	ň	0	0(0) 0)
17	26.4	69.7	ő	ő	0	ň	0	0	ō	-140	0	-440	-250	-0			0	0(0)
10	25.9	71.4	ő	97	o o	ŏ	ő	ŏ	ŏ	-140	ő	-440	-250	-0	ŏ	ŏ	Ö	0 (0)
19	25.3	73.5	ō	125	0	ō	0	0	ō	-140	0	-440	-250	-0	0	0	0	0 (0)
20	24.0	75.2	ō	140	ō	ō	0	ō	ō	-140	Ö	-440	-250	-0	ō	ō.	ō	ōi	6)
21	24.4	76.5	0	166	0	0	0	0	0	-140	0	-440	-250	-0	0	0	0	0 (0)
2.2	24.0	77.0	0	105	0	0	0	0	0	-140	0	-440	-250	-0	0	0	0	0 (0)
2.3	23.6	79.2	0	203	0	0	0	0	0	-140	0	-440	-250	-0	0	0	0	0 (0)
24	23.2	00.6	0	222	0	0	0	0	0	-140	0	-440	-250	-0	0	0	0	0 (0)

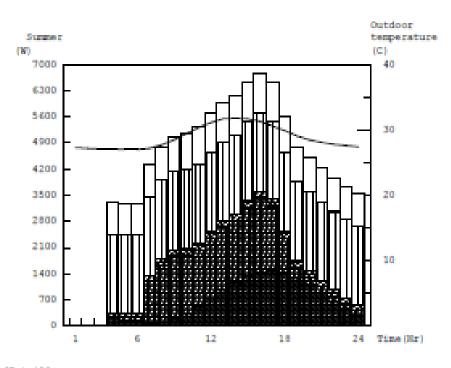
Total heat load in heating is not contained latent heat. If total indoor heat load is negative, it is not calculated.

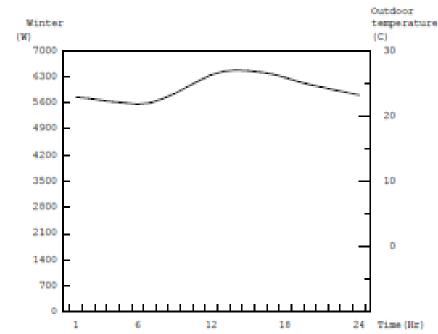
F/A : Fresh air SH : Sensible heat LH : Latent heat

Heat load graph

Room name	Floor	System	Rooms	Usage	Fl area(m2)	Height(m)	No of person
Living Room 1	1	1	1	Hotel	25.3	3.6	5







[Betail]

[Detail]		3 33 33 33 3	3 30 33 33 33	33 33 33 33	0.000.0000	12 27 12 27 1	2022 2022 C	********								
	Time	Outer	Roof	Inner	Floor	Window	Infilt	ration	Bunan	body	Light	Equip	ments	Fresh	Total	Selected heat
(W)		wall	Ceiling	wall	24004	W.LINDON	511	Ш	SII	LH	-ing	SH	LH	air	load	load
Sumer	16	507	753	164	18	1965	32	158	295	430	880	500	0	1042	6744	7081
Winter	6	15	286	6	1	19	2	0	-148	0	-440	-250	-0	7	7	6



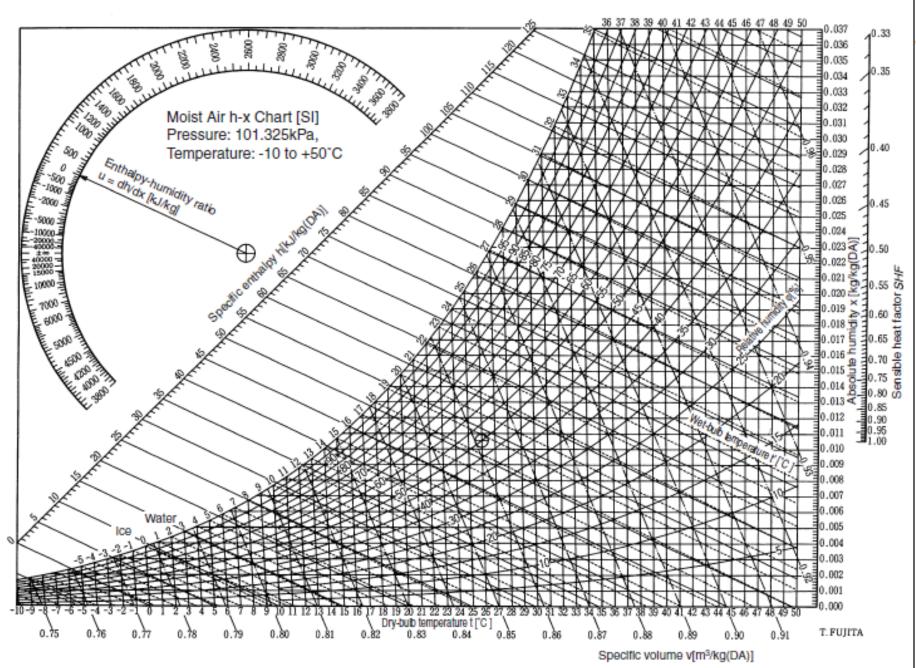
Any Question?



Psychometric Chart

How to use the Psychometric Chart

- Mixture of Air -

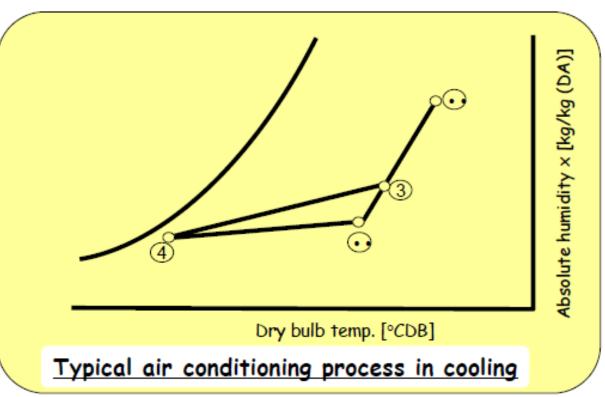


Introduction



◆ The psychrometric chart allows for:

Instant understanding of air properties; Display of change of the state of the air; Display of the air conditioning process.





Contents



- 1. What the composes the Psychometric Chart
- Mixture of Air
- 3. Summary

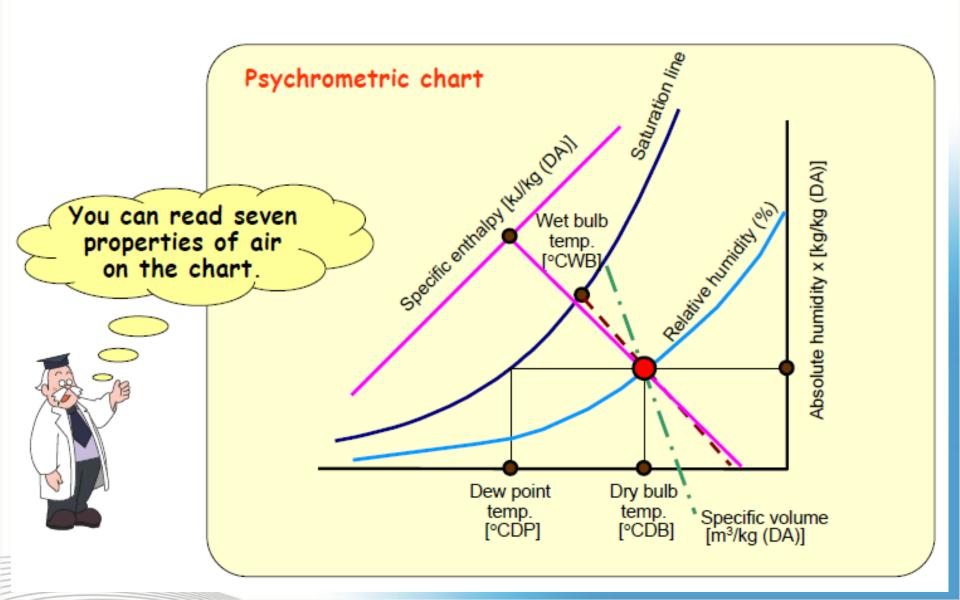
Air conditioners conduct various processes including mixing, cooling, heating, dehumidifying and humidifying of air.

Let's use a psychrometric chart to think about mixture of air.





1. What Compose the Psychrometric Chart Air Conditioning





1. Change Air State

Changes in Air State

- Heating
- Cooling
- Cooling/Dehumidification
- Humidification

Air conditioners conduct various processes including mixing, cooling, heating, dehumidifying and humidifying of air.

Let's use a psychrometric chart to think about mixture of air.



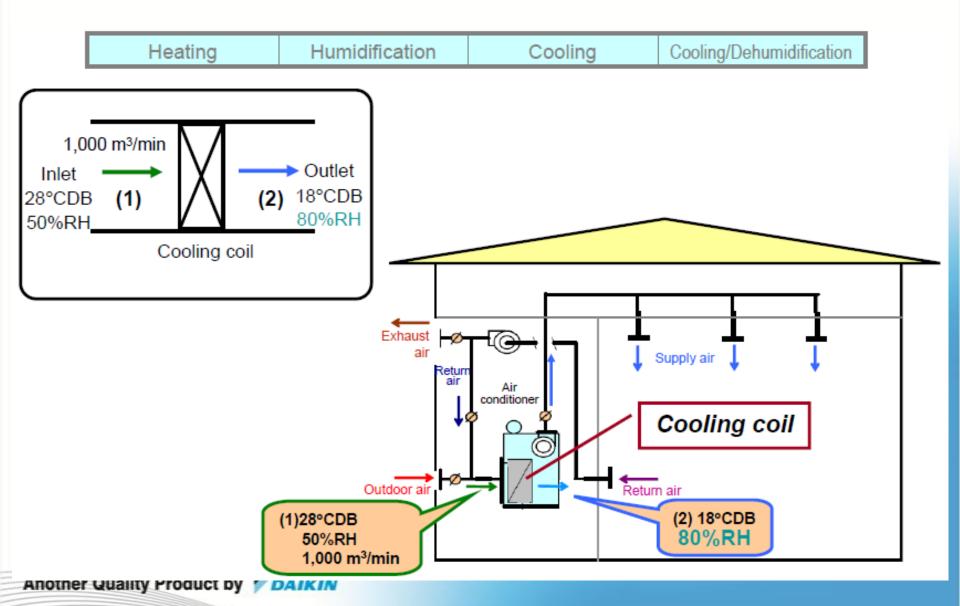


Heating Humidification Cooling Cooling/Dehumidification 1000 m³/min Inlet Outlet 28°CDB 18°CDB (2) (1) 50%RH Cooling coil (Surface temperature of 17°C) Exhaust air 👃 Supply air 🌡 Retur Air conditioner Cooling coil Outdoor air Return air (1)28°CDB (2) 18°CDB 50%RH 1,000m3/min



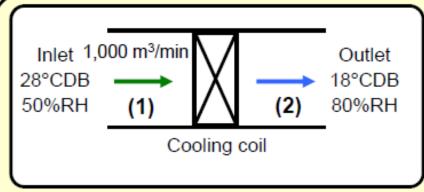
Heating Humidification Cooling/Dehumidification Cooling Specific enthalpy [kJ/kg] [DAN] Absolute humidity × [kg/kg (DA)] Inlet 1,000 m³/min Outlet 28°CDB 18°CDB 50%RH (2) (1)Cooling coil (Surface temperature of 17°C) 91.6% RH 50° 0 RH Amount of cooling generated by the coil 0.0118 (1)(2) $10.27 \times \frac{1,000}{0.87}$ • •11,805 kJ/min = 197 kJ/s18 28 = 197 kW 16.6 Dry bulb temp. [°CDB] 61 Dew point temperature of the coil inlet air is 16.6°C Specific volume [m³/kg(DA)]







Heating Humidification Cooling Cooling/Dehumidification



Amount of cooling by the coil

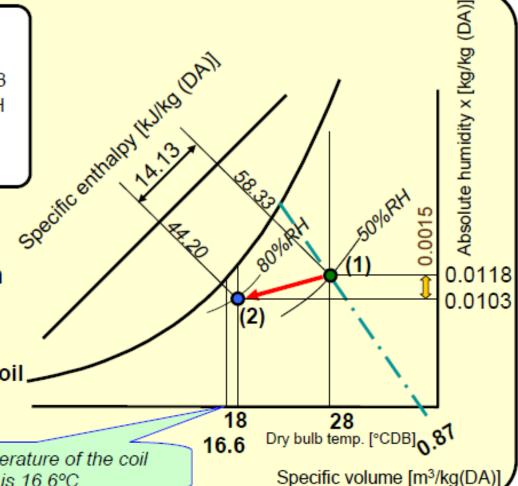
$$14.13 \times \frac{1,000}{0.87} = 16,241 \text{ kJ/min}$$

$$= 271 \text{ kJ/s} = 271 \text{kW}$$

Amount of dehumidification by the coil,

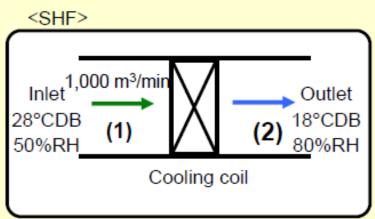
$$0.0015 \cdot \frac{1,000}{0.87} = 1.72 \text{ kg/min}$$

Dew point temperature of the coil inlet air is 16.6°C



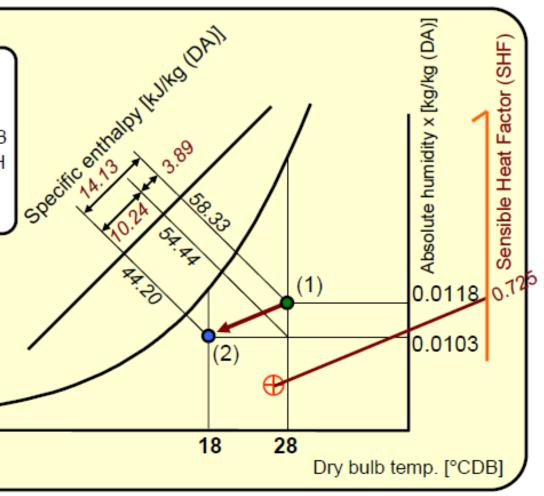


Heating	Humidification	Cooling	Cooling/Dehumidification
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SHF (Sensible Heat Factor)

$$= \frac{\text{Sensible heat}}{\text{Total heat}} = \frac{10.24}{14.13} = 0.725$$





3 Method of Cooling Load

- 1. Heat Load Calculation
- 2. ACH (Air Change per Hour)
- 3. Assuming method..... Ex 1 $m^2 = 700$ btu/h

Calculate Heat Load with ACH

- Room Dimension (L x W x H)= $13 \times 10 \times 6 = 780 \text{ m}^3$
- Ambient Temperature = 30 °C, RH = 79%
- Target Room Condition = 24 °C, RH = 50%
- ACH = 15 kali
- Fresh Air Assume 10%

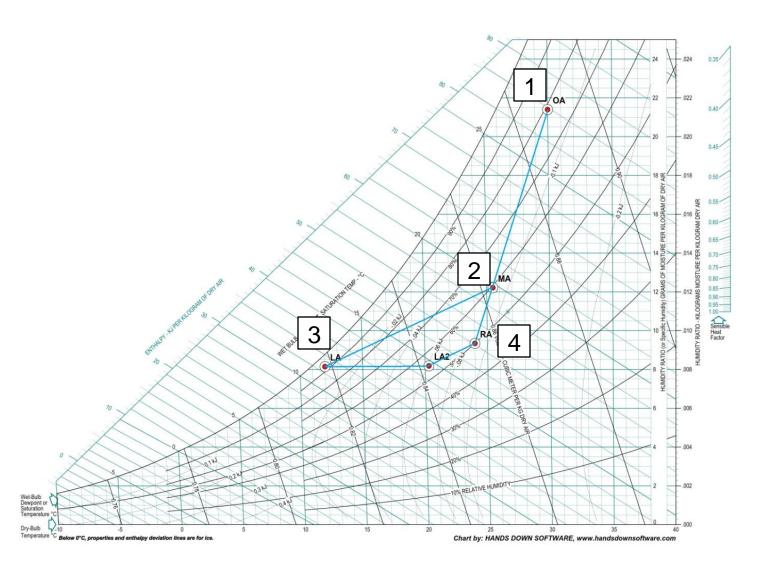


DETERMINE ????

- 1. Airflow = 15 ACH x 780 m3 = 11700 CMH 6886 CFM
- 2. Fresh Air = 10% x Airflow = 1170 CMH 688.6 CFM
- 3. Mixing Air (Tma) = 25.5° C, RH = 59.5%

4. Leaving Air Condition $T_{LA} = 12^{\circ}C$ RH = 93%







Calculate Cooling Load

- 1. $Q_{\text{sensible}} = 4.5 \times \text{CFM} \times (h_x h_{la})$ 183,753 Btu/h or 53.85 kW or 15.31 TR
- 2. $Q_{Laten} = 4.5 \times CFM \times (h_{mix} h_x)$ 137,582 Btu/h or 40.32 kW or 11.47 TR
- 3. $Q_{Total} = 4.5 \times CFM \times (h_{mix} h_{la})$ 321,335 Btu/h or 94.17 kW or 26.7 TR
- 4. Sensible Heat Ratio = 0.572
- 5. Air Condition After Reheat = 20.3°C RH =55%
- 3. $Q_{Reheat} = 4.5 \times CFM \times (h_{la2} h_{la})$ 114.032 Btu/h or 33.42 kW



Conclusion

- 1. Cooling Capacity = 94.2 kW
- 2. Capacity Reheat = 33.4 kW
- 3. Airflow = 11700 CMH or 6886 cfm
- 4. Room is class 100,000 should be complete with Medium and HEPA filter



Thank You